inwardly in the axial direction and outwardly in the radial direction with respect to the first groove **12**. Located in the groove **12** is a sealing element **14**, with a first axial sealing portion **15** placed securely onto the shoulder **1** and onto the stop ring **11**. The second sealing portion **16** branches off from the first sealing portion **15** radially outwardly into the second groove **13** at its inner axial end in the form of a sealing lip. The sealing lip **16** deviates outward by a tilt angle β with respect to a perpendicular median so that the second groove **13** is divided into two partial spaces **20** and **21** that can be used as additional lubricant spaces. The sealing lip **16** forms a gap seal with the wall of groove **13** so that the penetration of dirt into the bearing interior is made more difficult.

Opposite the sealing lip **16**, the third sealing portion **17** extends radially inwardly. It extends axially outwardly, offset by a tilt angle a with respect to a perpendicular median so that portion **17** sits against shoulder **1** and against the stop ring **11** only at a point. As can be seen from FIGS. **2** and **3**, the third sealing portion **17** of the sealing element is provided with slots **19** evenly spaced from one another around it. On one hand, these slots act as lubricant reservoirs and on the other hand enable unhindered lubricant flow through the bearing. This is done by passing lubricant into the track area **6** from the outside through axial grease bore **4** and radial grease bore **5**. The lubricant is then able to exit the bearing through the slots **19** of the third sealing portion **17**, through the labyrinthine seal between sealing lip **16** and notch **13**, as well as through the gap between the outer race **10** and the first sealing portion **15**. The first axial sealing portion **15** of the sealing element **14** is designed such that it nearly takes up all of the first groove **12** completely so that only a small gap is formed, thereby making the penetration of contaminants from the outside into the bearing interior more difficult.

In FIG. **4**, the guide roller is shown with outer race **10** shifted from its neutral position toward the right in the direction of the arrow. As the outer race rotates, points of contact **22** and **23** are formed between sealing section **16** and the wall of the second groove **13** on the left side as well as between the transition area between the second and third sealing portions **16** and **17** and the opposite face of the outer race **10** on the right side. In this way, a frictional seal is created. If the outer race **10** moves in the direction opposite to the arrow, a friction seal again arises that in that case is formed by points of contact **24** and **25**.

References

1 Shoulder
2 Pin
3 Center Section
4 Axial grease bore
5 Radial grease bore
6 Inner track
7 Cage
8 Bearing needles
9 Outer track
10 Outer race
11 Stop ring

-continued

References

12 First groove
13 Second groove
14 Sealing element
15 First axial sealing section
16 Second sealing section, extending radially outwardly
17 Third sealing section, extending radially inwardly
18 Receiving opening
19 Slot
20 Partial space
21 Partial space
22 Point of contact
23 Point of contact
24 Point of contact
25 Point of contact
α, β Tilt angles

What is claimed is:

1. A guide roller with at least one roller body set and one inner race in the form of a pin **(2)** that is located inside of a receiving opening **(18)** of an outer race **(10)**, whereby the inner race contains an axial flange at each outer end which is located inside a first groove **(12)** of the outer race **(10)** so that an axial shift of the outer race **(10)** with respect to the inner race is limited and whereby in the first groove **(12)**, another groove **(13)** is present at each end that extends axially inwardly and radially outwardly and that contains a sealing element, the sealing element **(14)** is formed in one piece in a T-shape with a first axial sealing portion **(15)** located securely on the shoulder, a second sealing portion **(16)** that extends radially outwardly is located within the second groove **(13)** and a third sealing portion **(17)** that extends radially inwardly branches off from the first sealing portion **(15)** at an inner end thereof so that axial contact of the shoulder and the outer race **(10)** is prevented by the third sealing portion **(17)**.

2. A guide roller according to claim **1**, wherein the third sealing portion **(17)** extends in a direction that deviates radially from a perpendicular median by an angle α.

3. A guide roller according to claim **1**, wherein the third sealing portion **(17)** is provided with evenly spaced slots **(19)**.

4. A guide roller according to claim **1**, wherein the second sealing portion **(16)** extends in a direction that deviates radially from a perpendicular median by an angle β.

5. A guide roller according to claim **1**, wherein the flange is formed by a shoulder **(1)** of the pin **(2)** on one side and by a stop ring **(11)** on the other side.

6. A guide roller according to claim **1**, wherein the roller body set comprises bearing needles **(8)** guided by a cage **(7)**.

7. A guide roller according to claim **1**, wherein the pin **(2)** has an axially extending lubricant bore **(4)** with a radially extending lubricant bore **(5)** that branches therefrom, **(5)**, which opens into a track area **(6)** of the roller body set.

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